

* Dynamic Fluids

→ study of motion of fluid flow along with the force causing the flow.

→ Let us consider fluid is flowing in x-direction.

$$F = m \cdot a$$

$$\therefore \boxed{F_x = m \cdot a_x}$$

- ① $F_g \rightarrow$ (gravity force)
- ② $F_p \rightarrow$ (pressure force)
- ③ $F_v \rightarrow$ (viscous force)
- ④ $F_T \rightarrow$ (force due to turbulence)
- ⑤ $F_c \rightarrow$ (force due to compressibility)

$$\therefore \boxed{F_{\text{net}} = (F_x)_g + (F_x)_p + (F_x)_v + (F_x)_T + (F_x)_c}$$

① If we neglected force due to compressibility.

$$\therefore \boxed{(F_x)_{\text{net}} = (F_x)_g + (F_x)_p + (F_x)_v + (F_x)_T}$$

→ Reynolds equation

② If $(F_x)_c$ & $(F_x)_T$ are neglected.

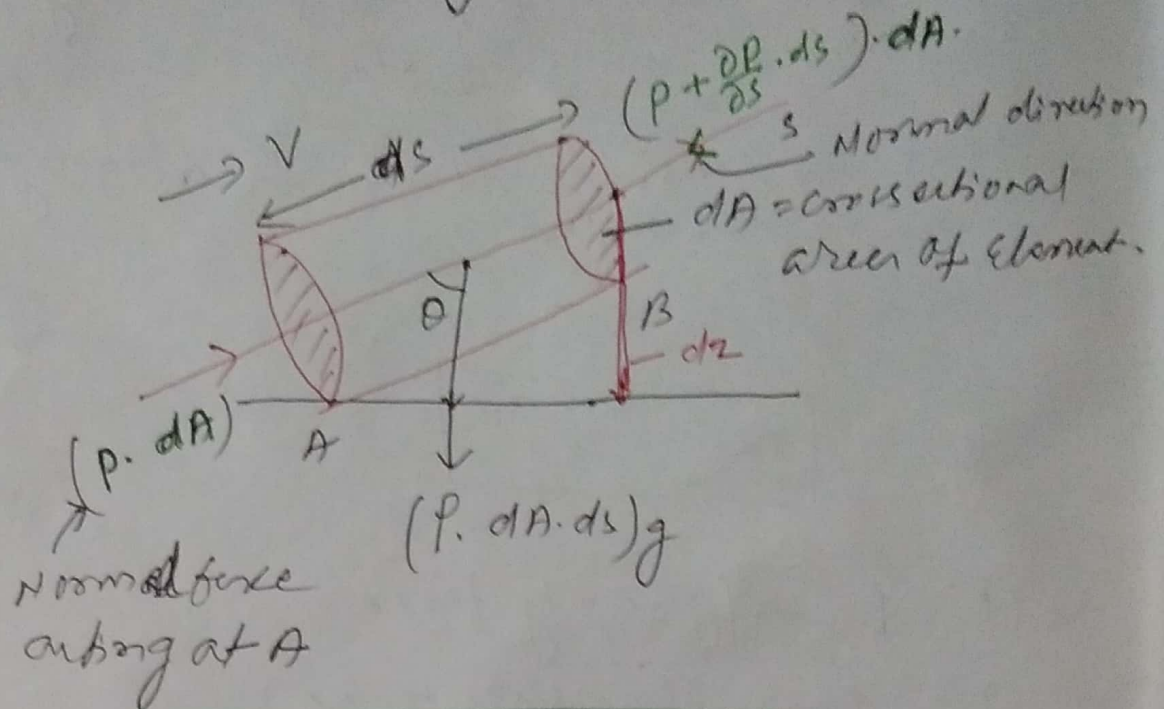
$$\therefore \boxed{F_x = (F_x)_g + (F_x)_p + (F_x)_v} \rightarrow \text{Navier Stokes equation}$$

$$\text{③ If } \boxed{F_x = (F_x)_g + (F_x)_p} \rightarrow \text{Euler's equation.}$$

* Euler's equation of motion

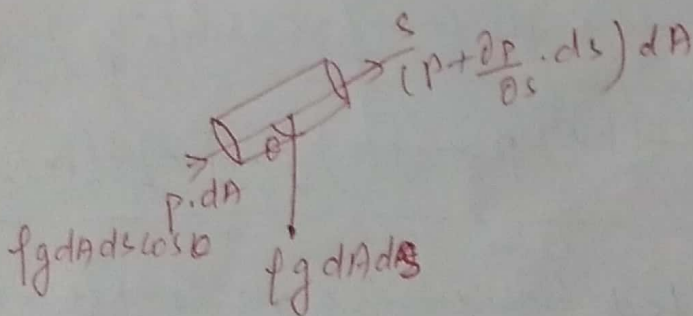
(2)

→ The Resultant force will be equal to the sum of gravity force & pressure force



$$\rightarrow \boxed{P_s + ds = P + \frac{\partial P}{\partial s} \cdot ds} \rightarrow \text{Taylor's equation.}$$

→ Let us consider density of fluid = ρ .



(1) Elements is moving along stream line

→ Using Newton's 2nd law

$$F_s = m \cdot a_s$$

$$\rightarrow P \cdot dA - \left(P + \frac{\partial P}{\partial s} ds \right) dA - \rho g dA ds \cos \theta = (\rho dA ds) \times a_s$$

$$\rightarrow P \cdot dA - P \cdot dA + \frac{\partial P}{\partial s} ds \cdot dA - \rho g dA ds \cos \theta = (\rho dA \cdot ds) \times a_s$$

$$\text{OR} \quad -\frac{\partial P}{\partial s} dA \cdot ds - \rho g dA ds \cos \theta = \rho dA ds \left\{ \frac{\partial v}{\partial t} + \underbrace{v \cdot \frac{\partial v}{\partial s}}_{\substack{\text{local} \\ \text{acceleration}}} \right\}$$

③

↓
Convective acceleration

$$\rightarrow -\frac{\partial P}{\partial s} dA \cdot ds - \rho g dA ds \cos \theta = \rho dA ds \left\{ \frac{\partial v}{\partial t} + v \cdot \frac{\partial v}{\partial s} \right\}$$

② If flow is steady

→ local acceleration will become zero.

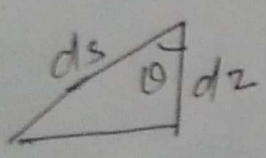
$$\boxed{\frac{\partial v}{\partial t} = 0}$$

→ Substituting equation ① with $\rho dA \cdot ds$

$$\rightarrow -\frac{\partial P}{\partial s} - g \cos \theta = v \cdot \frac{\partial v}{\partial s}$$

$$\rightarrow -\frac{\partial P}{\partial s} - g \cos \theta - v \cdot \frac{\partial v}{\partial s} = 0$$

$$\rightarrow \boxed{\frac{\partial P}{\partial s} + g \cos \theta + v \cdot \frac{\partial v}{\partial s} = 0}$$



$$\cos \theta = \frac{dz}{ds}$$

$$\therefore \frac{\partial P}{\partial s} + g \frac{dz}{ds} + v \cdot \frac{\partial v}{\partial s} = 0$$

$$\rightarrow \frac{dP}{\rho \cdot ds} + g \cdot \frac{dz}{ds} + v \cdot \frac{dv}{ds} = 0$$

$$\rightarrow \boxed{\frac{dP}{\rho} + g \cdot dz + v dv = 0}$$

→ Euler's Equation of motion

* important point for Euler's equations

(4)

(i) F_v = force due to viscosity is neglected,
 F_c is also negligible.

→ So, this equation is valid for non-viscous
or inviscid fluid flows.
↓
Ideal fluid flow.

(ii) $\frac{\partial v}{\partial t} = 0$ → for steady flow.

→ It is valid for steady flows.

(iii) F_T is neglected.

→ flow is laminar or stream line flow.

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* Bernoulli's equation

$$\rightarrow \boxed{\frac{dp}{\rho} + g dz + v \cdot dv = 0} \rightarrow \text{Euler's Equation}$$

Integrating both side

$$\rightarrow \int \frac{dp}{\rho} + \int g dz + \int v \cdot dv = \int (0)$$

$$\rightarrow \boxed{\rho = \text{constant}} \rightarrow \text{fluid is incompressible}$$

$$\rightarrow \boxed{\frac{p}{\rho} + gz + \frac{v^2}{2} = \text{constant}}$$

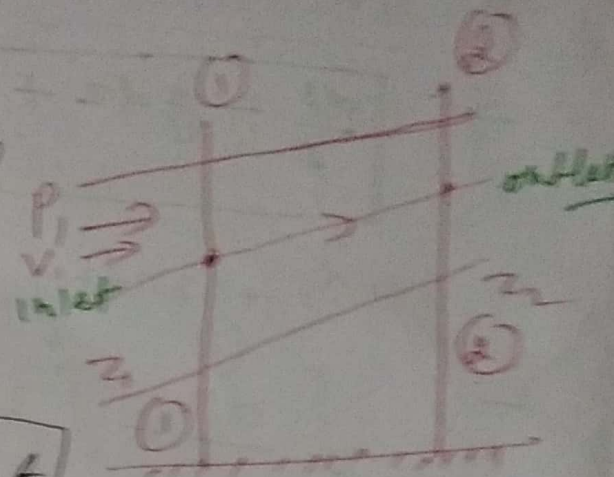
$$\rightarrow \frac{p}{\rho g} + z + \frac{v^2}{2g} = \frac{\text{const.}}{g} = \text{constant.}$$

$$\rightarrow \boxed{\frac{p}{\rho g} + z + \frac{v^2}{2g} = C} \rightarrow \text{Bernoulli's equation.}$$

$\rightarrow$ If flow is rotational

* Bernoulli's Equation

- ① Flow is inviscid & incompressible (ideal fluid)
- ② steady flow.
- ③ irrotational flow.



$$\rightarrow \left[\frac{P}{\rho g} + \frac{v^2}{2g} + z = \text{constant} \right]$$

$$\rightarrow \left[\frac{P_1}{\rho g} + \frac{v_1^2}{2g} + z_1 = \frac{P_2}{\rho g} + \frac{v_2^2}{2g} + z_2 \right]$$

→ Bernoulli's equation for an ideal fluid.

$$\rightarrow \frac{P}{\rho g} + \frac{v^2}{2g} + z = \text{constant}$$

$\downarrow$ $\downarrow$ $\downarrow$
 ① ② ③

terms ③ potential Energy per unit weight
 $\downarrow$
 head.

or
potential head

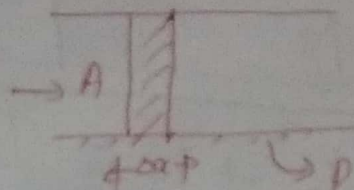
$$= \frac{\rho g \cdot z}{\rho g} = z = \text{head}$$

$\downarrow$

terms ② $\frac{v^2}{2g}$ = kinetic Energy per unit weight

terms (3)

$$\frac{P}{\rho g}$$



pressure constant for small Δx (displacement)

- force required to maintain the flow = $P \cdot A$
- work done by pressure to maintain the flow.

$$W = P \cdot A \cdot \Delta x$$

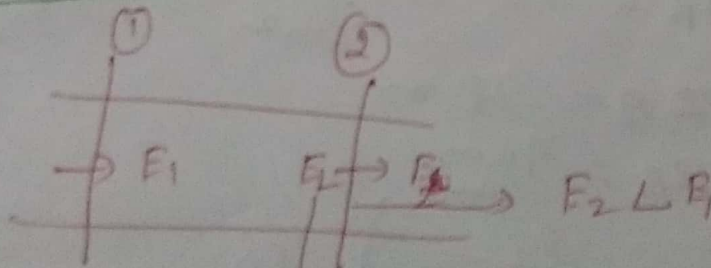
$$\begin{aligned} \rightarrow \text{Weight of elements} &= P \cdot A \cdot \Delta x \cdot g \\ &= \frac{W}{P \cdot A \cdot \Delta x \cdot g} = \frac{P \cdot A \cdot \Delta x}{P \cdot A \cdot \Delta x \cdot g} \\ &= \frac{P}{\rho g} \end{aligned}$$

$$\rightarrow \frac{P}{\rho g} = \text{work required to maintain the flow per unit weight}$$

$$\rightarrow \frac{P}{\rho g} = \text{flow energy.}$$

→ Bernoulli's equation is conservation of energy.

* Bernoulli's equation for Real fluids



F_L = energy overcome the friction

→ Energy lost per unit weight = Head loss

$$\rightarrow \left[\frac{E_1}{\rho g} = \frac{E_2}{\rho g} + h_L \right]$$

$$\rightarrow \left[\frac{P_1}{\rho g} + \frac{V_1^2}{2g} + z_1 = \frac{P_2}{\rho g} + \frac{V_2^2}{2g} + z_2 + h_L \right]$$

$$\otimes \left[(E)_{\text{inlet}} > (E)_{\text{outlet}} \right]$$

$$\rightarrow \left[\frac{E_{\text{inlet}} - E_{\text{out}}}{\rho g} = h_L \right]$$

Q A pipe of diameter 400 mm carries water at a velocity of 25 m/s, pressure at point A & B is $29.43 \frac{\text{N}}{\text{cm}^2}$ and $22.563 \frac{\text{N}}{\text{cm}^2}$ respectively. While datum head at A & B are 28 m & 30 m find Head loss between A & B

Soln

Given

$$D = 400 \text{ mm} = 0.4 \text{ m}$$

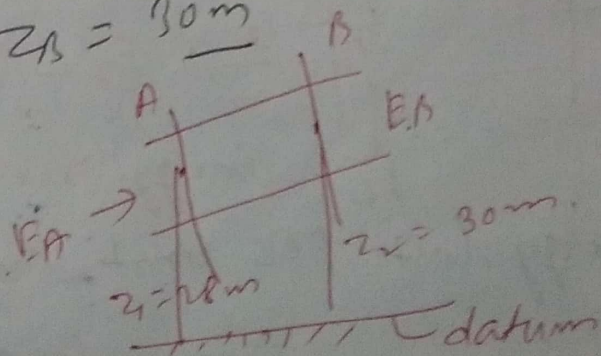
$$V = 25 \text{ m/s}$$

$$P_A = 29.43 \frac{\text{N}}{\text{cm}^2} = 29.43 \times 10^4 \frac{\text{N}}{\text{m}^2}$$

$$P_B = 22.563 \frac{\text{N}}{\text{cm}^2} = 22.563 \times 10^4 \frac{\text{N}}{\text{m}^2}$$

$$Z_A = 28 \text{ m}$$

$$Z_B = 30 \text{ m}$$



→ for incompressible fluid

$$A_1 V_1 = A_2 V_2 \quad \text{Continuity equation}$$

$$A_1 = A_2 \quad \therefore V_1 = V_2 = 25 \text{ m/sec.}$$

$$\rightarrow \left[\left(\frac{V^2}{2g} \right)_A = \left(\frac{V^2}{2g} \right)_B \right]$$

$$\rightarrow \frac{P_A}{\rho} + \frac{V_A^2}{2g} + Z_A = \frac{P_B}{\rho} + \frac{V_B^2}{2g} + Z_B + h_L$$

$$\rightarrow \frac{29.43 \times 10^4}{1000 \times 9.81} + \frac{25^2}{2 \times 9.81} + 28 = \frac{22.563}{1000 \times 9.81} + \frac{25^2}{2 \times 9.81} + 30 + h_L$$

$$\therefore \boxed{h_L = 5 \text{ m}}$$